

Vibrations and Waves – Tutorial Questions VnW1 ANSWERS

ANSWER TO QUESTION

(i) $[k] = [\tau/\theta] = \text{N m/rad}$

(ii) $\tau = I\ddot{\theta}$ (Compare with $F = m\ddot{x}$.)

$$-k\theta = I\ddot{\theta} \Rightarrow \ddot{\theta} + (k/I)\theta = 0$$

(Note this is the same as a mass on a spring with $m \rightarrow I$ and $x \rightarrow \theta$.)

(iii) Differentiating, $\theta(t) = A \cos(\omega_0 t + \phi)$ twice gives $\ddot{\theta} = -\omega_0^2 A \cos(\omega_0 t + \phi)$. Substituting into the equation of motion gives

$$-\omega_0^2 A \cos(\omega_0 t + \phi) + (k/I)A \cos(\omega_0 t + \phi) = 0$$

Cancelling $A \cos(\omega_0 t + \phi)$, leaves $\omega_0^2 = k/I$ — i.e., trial solution is valid if $\omega_0^2 = k/I$. Furthermore, it has two independently adjustable constants as expected for the general solution of a 2nd-order differential equation.

Similarly for $\tilde{\theta}(t) = A \exp(i\omega_0 t + \phi)$, differentiating gives $\ddot{\tilde{\theta}} = -\omega_0^2 A \exp(i\omega_0 t + \phi)$. Substituting into the equation of motion and cancelling yields the same result, $\omega_0^2 = k/I$.

Hence $\omega_0 = \sqrt{k/I}$ and $T = 2\pi/\omega_0 = 2\pi\sqrt{I/k}$.

(iv) From (iii), $T = 2\pi\sqrt{I/k}$. Rearranging and substituting for I gives

$$k = I \frac{4\pi^2}{T^2} = mR^2 \frac{4\pi^2}{T^2} = \frac{4\pi^2 mR^2}{T^2} = \frac{4\pi^2 (40 \times 10^{-6} \text{ kg})(4.5 \times 10^{-3} \text{ m})^2}{(1/3 \text{ s})^2} = 2.88 \times 10^{-7} \text{ Nm/rad}$$

(v) (a) Initial conditions $\theta(0) = 4.5 \text{ rad}$ and $\dot{\theta}(0) = 0$. Sub into general solution, gives

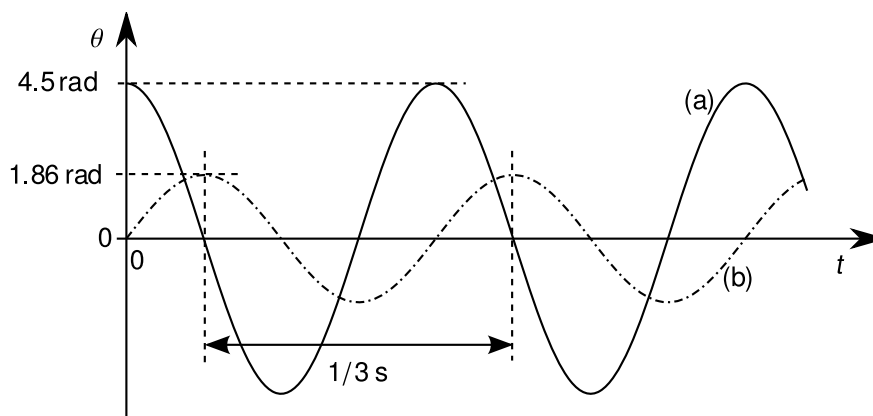
$$\theta(0) = A \cos \phi = 4.5 \text{ rad}, \quad \dot{\theta}(0) = -\omega_0 A \sin \phi = 0 \Rightarrow A = 4.5 \text{ rad}, \quad \phi = 0$$

$$\left(\text{more generally, can use: } A^2 = \theta^2 + \dot{\theta}^2 / \omega_0^2, \quad \tan \phi = \frac{-\dot{\theta}(0)}{\omega_0 \theta(0)} \right)$$

(b) Initial conditions $\theta(0) = 0$ and $\dot{\theta}(0) = 35.0 \text{ rad/s}$.

$$\theta(0) = A \cos \phi = 0, \quad \dot{\theta}(0) = -\omega_0 A \sin \phi = 35.0 \text{ rad/s}$$

$$\Rightarrow A = 35.0 \text{ rad} / \omega_0 = \frac{35.0}{2\pi \times 3} = 1.86 \text{ rad}, \quad \phi = -\pi/2$$



(vi) (a) Using $x \rightarrow \theta$ and $m \rightarrow I$,

$$PE = \frac{1}{2}k[x(t)]^2 \rightarrow PE_{\text{rot}} = \frac{1}{2}k[\theta(t)]^2$$

$$KE = \frac{1}{2}m[v(t)]^2 \rightarrow KE_{\text{ang}} = \frac{1}{2}I[\dot{\theta}(t)]^2$$

$$E = PE_{\text{rot}} + KE_{\text{ang}} = \frac{1}{2}(k\theta^2 + I\dot{\theta}^2)$$

(b) Differentiating total energy E , which is conserved, w.r.t. t gives

$$k\theta\dot{\theta} + I\dot{\theta}\ddot{\theta} = 0 \Rightarrow k\theta + I\ddot{\theta} = 0$$

TOPICS FOR DISCUSSION

It is essential that the students understand the following topics early in this course.

Topic 1

Prove that

$$x(t) = A \sin(\omega t + \theta)$$

and

$$x(t) = B \cos \omega t + C \sin \omega t$$

are both mathematically equivalent to $x(t) = A \cos(\omega t + \phi)$, where θ , B , and C are constants. Write down expressions relating ϕ to θ , A to B and C , and ϕ to B and C .

Topic 2: Using complex variables

See Sections 1.4 and 1.5 of my notes (on Blackboard) on circular motion and motion in the complex plane.

Possible exercise (from Section 1.5 of notes):

Show that if $\tilde{x}(t) = x(t) + iy(t)$ is a solution to the differential equation

$$a \frac{d^2 \tilde{x}}{dt^2} + b \frac{d\tilde{x}}{dt} + c\tilde{x} = f(t)$$

where a , b , c , $x(t)$, and $y(t)$ are all real valued, then

$$a \frac{d^2 x}{dt^2} + b \frac{dx}{dt} + cx = \text{Re}(f(t))$$

and

$$a \frac{d^2 y}{dt^2} + b \frac{dy}{dt} + cy = \text{Im}(f(t))$$

where $\text{Re}()$ and $\text{Im}()$ refer to the real and imaginary parts, respectively.

Topic 3: Taylor series expansion (1-D and 2-D)

The following question used to be on the tutorial sheet. I've removed it as a question because I've covered it in detail in notes and lectures, but it is very important that they understand Taylor series and linear response.

Q:

Show that the restoring force about any stable equilibrium point is linear (i.e. Hooke's Law is valid) for sufficiently small displacements. Why is this important? What happens for larger displacements?

A:

Taylor series expansion of $V(x)$ about x_0 gives

$$V(x) = V(x_0) + V'(x_0)(x - x_0) + (1/2)V''(x_0)(x - x_0)^2 + (1/3!)V'''(x_0)(x - x_0)^3 + \dots$$

Because x_0 is a stable equilibrium, $V'(x_0) = 0$ (no net force) and $V''(x_0) > 0$ (local minimum of V .) Therefore the force, $F(x) = -dV(x)/dx$ is given by

$$F(x) = -V''(x_0)(x - x_0) - (1/2)V'''(x_0)(x - x_0)^2 - \dots$$

For small displacements $(x - x_0)$ we can neglect the second and higher terms, so that $F(x) = -V''(x_0)(x - x_0) = -k(x - x_0)$, where $k = V''(x_0) > 0$ i.e., a linear restoring force.

Note, small displacements are such that $|[(1/2)V'''(x_0)/V''(x_0)](x - x_0)| \ll 1$ (with similar relationships for the higher order terms.)

This is an important point: physical systems are not purely linear or purely non-linear, but have ranges in which they can be treated as linear and ranges in which they are non-linear. However, all behave linearly if the displacement is small enough. This is a consequence of $V(x)$ being a smooth function of x .

If you allow the displacements to increase, you should see that there comes a point when the second and higher terms that you have neglected here start to become relevant again and the system starts to become non-linear. Of course the size of the linear range can vary enormously between different systems.

In a non-linear system the oscillatory motion is no longer simple harmonic and it contains components with different frequencies.